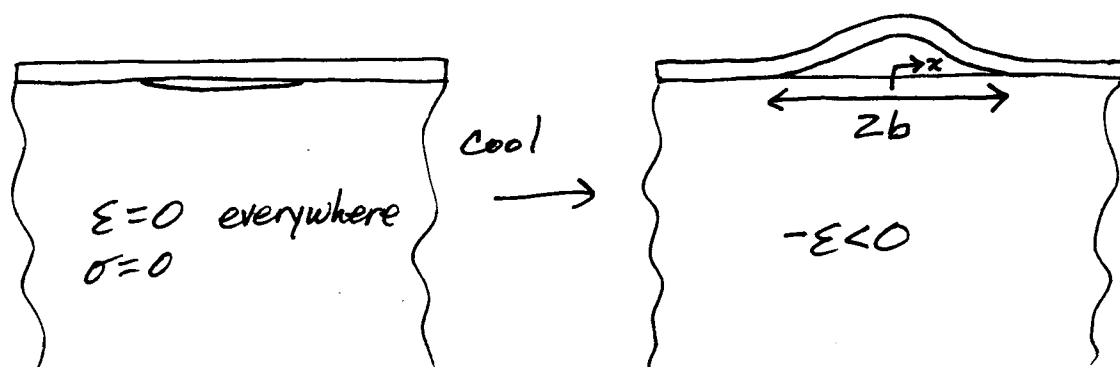


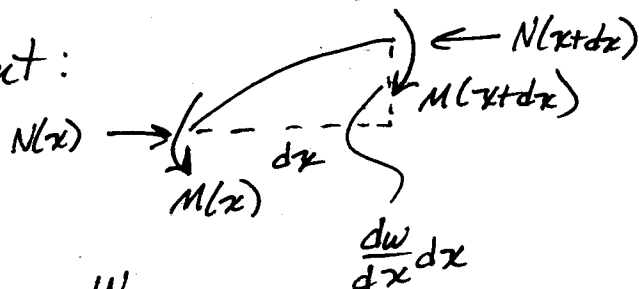
Buckle Delamination



What is the energy release rate?

First we need to analyze the buckled configuration.

Consider a differential element:



$$\sum F_x = N(x) - N(x+dx) = 0 \rightarrow \frac{dN}{dx} = 0$$

$$\rightarrow N = \text{constant}$$

$$\sum M_z^{(dx)} = N(x) \frac{dw}{dx} dx + M(x) - M(x+dx) = 0$$

$$\rightarrow -\frac{dM}{dx} + N \frac{dw}{dx} = 0$$

Sign convention for M and w $\rightarrow M = -DX = -D \frac{d^2 w}{dx^2}$

$$D = \frac{E k^3}{12(1-\nu^2)} = \text{Plane strain bending stiffness per unit thickness into page}$$

$$\rightarrow D \frac{d^3 w}{dx^3} + N \frac{dw}{dx} = 0$$

General solution symmetric about $x=0$

$$\rightarrow w = A + B \cos \sqrt{\frac{N}{D}} x \quad w(\pm b) = 0 \rightarrow B = A$$

$$w' = -B \sqrt{\frac{N}{D}} \sin \sqrt{\frac{N}{D}} x$$

$$w'(\pm b) = \mp B \sqrt{\frac{N}{D}} \sin \sqrt{\frac{N}{D}} b = 0$$

$$\rightarrow \sqrt{\frac{N}{D}} b = \pi \rightarrow \boxed{N = \frac{\pi^2 D}{b^2}}$$

This tells us the force in the buckled region.
To determine A we need to describe the axial strain of the centerline ϵ^A .

This mildly non-linear kinematics theory $\rightarrow \epsilon^A = \frac{du}{dx} + \frac{1}{2} \left(\frac{dw}{dx} \right)^2$

The film is bonded to the rigid substrate and
so $u(x=b) - u(x=-b) = -2b\epsilon$ (Note $\epsilon > 0$ for a contraction)

The axial material response of the film is linear elastic such that $\epsilon^A = \frac{\sigma^A}{E'} = \frac{N}{E'h}$

$$\rightarrow \epsilon^A = \frac{-N}{E'h} = \frac{du}{dx} + \frac{1}{2} \left(\frac{dw}{dx} \right)^2$$

$$\int_{-b}^b \frac{du}{dx} dx = u(b) - u(-b) = -2b\epsilon = \int_{-b}^b \frac{-N}{E'h} - \frac{1}{2} \left(\frac{dw}{dx} \right)^2 dx$$

$$-2b\varepsilon = \underbrace{\frac{-N}{E'h}(2b)}_{\varepsilon_c = \frac{\pi^2}{12} \left(\frac{h}{b}\right)^2} - \frac{1}{2} A^2 \frac{\pi^2}{b^2} \underbrace{\int_{-b}^b \sin^2\left(\frac{\pi x}{b}\right) dx}_b$$

$$A^2 = \frac{4b^2}{\pi^2} (\varepsilon - \varepsilon_c) \quad \varepsilon_c = \text{critical strain for buckling}$$

Recall: $w = A \left(1 + \cos \frac{\pi x}{b}\right)$

So now we know the solution for the buckled film. Let's compute \mathcal{G} .

$$U_{\text{unbuckled}} = \frac{1}{2} E' \varepsilon^2 h (L - 2b) \quad \text{where } L = \text{total length of film}$$

$$U_{\text{buckled}} = \int_{-b}^b \frac{D}{2} (w'')^2 + \frac{E'h}{2} (\varepsilon^A)^2 dx$$

$$= \frac{D}{2} A^2 \frac{\pi^4}{b^4} \underbrace{\int_{-b}^b \cos^2\left(\frac{\pi x}{b}\right) dx}_b + \frac{E'h}{2} \frac{\pi^4}{144} \frac{h^4}{b^4} (2b)$$

$$= \frac{E'h^3}{24} \frac{4b^2}{\pi^2} (\varepsilon - \varepsilon_c) \frac{\pi^4}{b^4} b + \frac{E'h}{2} \frac{\pi^4}{144} \frac{h^4}{b^4} (2b)$$

$$U = \frac{E'h}{2} \left[(L-2b) \varepsilon^2 + \frac{\pi^2 h^2}{3b} \varepsilon - \frac{\pi^4 h^4}{36 b^3} + \frac{\pi^4 h^4}{72 b^3} \right]$$

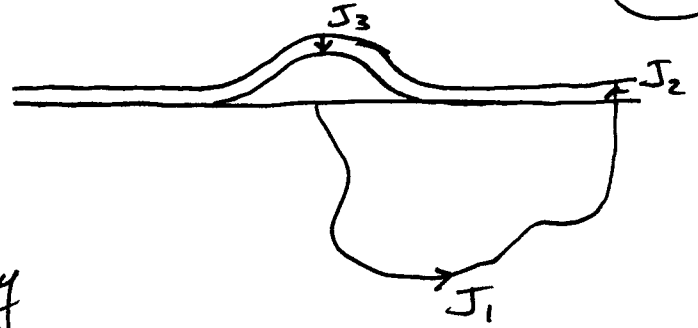
$$\quad \quad \quad \underbrace{-\frac{\pi^4 h^4}{72 b^3}}$$

$$\mathcal{G} = -\frac{\partial U}{\partial (2b)} = -\frac{1}{2} \frac{\partial U}{\partial b} = \frac{E'h}{2} \left[\varepsilon^2 + \frac{\pi^2 \varepsilon h^2}{6 b^2} - \frac{\pi^4 h^4}{48 b^4} \right]$$

$$\quad \quad \quad \underbrace{\quad}_{2\varepsilon\varepsilon_c} \quad \quad \quad \underbrace{\quad}_{3\varepsilon_c^2}$$

$$\boxed{\mathcal{G} = \frac{E'h}{2} (\varepsilon + 3\varepsilon_c)(\varepsilon - \varepsilon_c)}$$

Next, let's try J .



$$J_2 = \int_0^h W - \sigma_{xx} \epsilon_{xx} dy$$

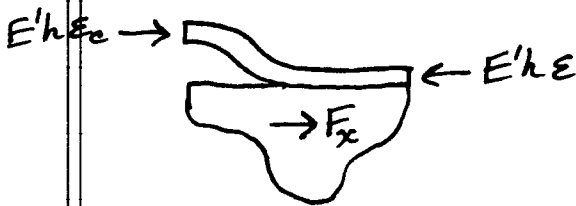
$$W = \frac{1}{2} \sigma_{xx} \epsilon_{xx} \rightarrow J_2 = -\frac{E'h}{2} \epsilon^2$$

$$\begin{aligned} J_3 &= \int_h^0 -W + \sigma_{xx} \epsilon_{xx} (-dy) \quad \text{note } \epsilon_{xx} \neq \epsilon^A \\ &= \int_0^h \frac{1}{2} \sigma_{xx} \epsilon_{xx} dy \quad \begin{aligned} \epsilon_{xx} &= \epsilon^A + \chi y \\ \text{again } W &= \frac{1}{2} \sigma_{xx} \epsilon_{xx} \end{aligned} \\ &= \frac{D}{2} (w'')^2 + \frac{E'h}{2} (\epsilon^A)^2 \quad \epsilon^A = \epsilon_c \text{ on } \Gamma_3 \end{aligned}$$

$$= \frac{E'h^3}{24} \frac{4b^2}{\pi^2} (\epsilon - \epsilon_c) \frac{\pi^4}{b^4} + \frac{E'h}{2} \epsilon_c^2 = \frac{E'h}{2} (4\epsilon\epsilon_c - 3\epsilon_c^2)$$

$$J_1 = \int_{\Gamma_1} W_{\text{rigid}} - t_x \epsilon_{xx}^{\text{rigid}} - t_y \epsilon_{xy}^{\text{rigid}} d\Gamma$$

$$J_1 = \epsilon \int_{\Gamma_1} t_x d\Gamma = \epsilon F_x = \frac{E'h}{2} (2\epsilon^2 - 2\epsilon\epsilon_c)$$



$$F_x = E'h\epsilon - E'h\epsilon_c$$

$$\begin{aligned} \rightarrow J &= J_1 + J_2 + J_3 = \frac{E'h}{2} [-\epsilon^2 + 4\epsilon\epsilon_c - 3\epsilon_c^2 + 2\epsilon^2 - 2\epsilon\epsilon_c] \\ &= \frac{E'h}{2} [\epsilon^2 + 2\epsilon\epsilon_c - 3\epsilon_c^2] \\ J &= G = \frac{E'h}{2} (\epsilon - \epsilon_c)(\epsilon + 3\epsilon_c) \end{aligned}$$